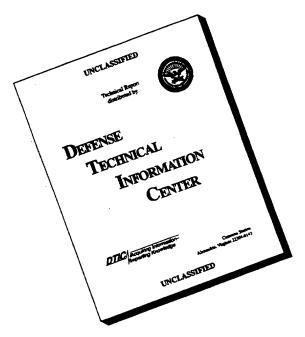
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TURBULENT HEAT TRANSFER INVESTIGATION: TURBULENCE LENGTH SCALES AND TURBINE HEAT TRANSFER



Jason Sharp Pete Harris

3 MAY 1996

FINAL REPORT 1 NOVEMBER 1995--9 JULY 1996

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Table of Contents

Abstract	i
Table of Contents	
List of Tables and Figures	
Nomenclature List	v
Introduction	
Theoretical Background	
Experimental Background	1
Experimental Methods	5
Experiment Setup	
Experiment Procedure	11
Uncertainty Analysis	12
Results and Discussion	
"Clean" Tunnel Configuration (No Turbulence Grid)	14
2 3/8 in Grid Configuration	16
1/2 in Grid Configuration	18
Overall Discussion	
Conclusions	20
Recommendations	
Acknowledgments	
References	22
Appendix A: Heat Transfer Graphs	23
Appendix B: Heat Transfer Data Sheets	
Appendix C: Length Scale Data Sheets	
Appendix D: Uncertainty Analysis	

List of Tables and Figures

Figure 1:	Taylor-Gortler Vortices	. 2
Figure 2:	Cascade Wind Tunnel Schematic	. 7
	Test Section Diagram	
	Traversable Test Probe	
Figure 5:	Test Blade Setup	. 8
	Grid of 2 3/8 in Diameter Bars	
Figure 7:	Gird of 1/2 in Diameter Bars	10
Figure 8:	Uncertainty Analysis	13
Figure 9:	Stagnation Point comparison	15
Figure 10	:Clean Tunnel Comparison	16
Figure 11	:Heat Transfer Comparison	1 <i>7</i>
Figure 12	Grid Comparison	19
Table 1:	List of Equipment	10
Table 2:	Clean Tunnel Flow Comparison	14
Table 3:	23/8 in Grid Comparison	16
	1/2 in Grid Comparison	18

Nomenclature List

B₁ - air inlet angle, degrees
B₂ - air exit angle, degrees
B_x - axial chord length, in
C - degrees Celsius

e - voltage

e₀ - zero velocity voltage

h - heat transfer coefficient, W/m²K

Hz - Hertz

I - electric current, amps

in - inches

K - degrees Kelvin

- thermal coefficient for air, W/mK

kg - kilogram

l - length of gold sheet, in

m - meters

N - Newton, kg m/s²
Nu - Nusselt number
Ohm - measure of resistance
Ohm/SQ - resistance per square

p - pitch distance between turbine blades, in

 P_{∞} - atmospheric pressure, psi psi - pounds per square inch q" - heat transfer flux, W/m²

q_e" - convective heat transfer flux, W/m² q_L" - conductive heat transfer flux, W/m₂

R(T) - autocorrelation function

R_{35.7}" - resistance per square of gold at 35.7° C, Ohm/SQ

Re - Reynolds number

s - seconds

S - surface arc length, in St - Stanton number

 T_{∞} - free-stream air temperature

T_{LC} - surface temperature of yellow liquid crystal band, C

torr - pressure measurement also mm Hg

T_u - longitudinal turbulence intensity (See Equation ?)

u - local velocity fluctuation

U - local velocity, m/s

W - Watts

w - width of gold film, in

x, y, z - longitudinal, lateral, vertical axis/distances, in

ρ - density, kg/m³
 ε - emmissivity

σ - Stefan-Boltzman constant, W/m²K⁴

μ - viscosity of air, Ns/m²

ΔΡ	- dynamic pressure, torr
Λ_{u}	- macro or integral turbulent length scale, m
λ_{u}	- microscale turbulent length scale, m
δ_{x}	- experimental uncertainty in parameter x

Introduction

Theoretical Background

The heat transfer on turbine blades directly affects the way engine designers can develop new turbines to allow for higher turbine inlet temperatures in engines. A better understanding of turbine blade heat transfer allows for better and more efficient cooling techniques to be developed. With more efficient cooling, increases in the turbine inlet temperature can be made without advances in materials technology. Since this has a direct and beneficial effect on the engine cycle design, reflected in greater specific thrust and lower thrust specific fuel consumption, this area is of great interest to the gas turbine engine industry. If a greater understanding of the heat transfer on such blades can be reached, new and better designs can be made.

The effect of turbulence on heat transfer has been known for some time but has yet to be physically modeled with great success. There are several parameters to describe the turbulent flow that are used in this study. The first of which is turbulence intensity. Turbulence intensity in the axial direction, noted as T_u , is one of the parameters of interest in this investigation.

Turbulence intensity is a method of measuring non-dimensional turbulence in a single direction. It is the fluctuating velocity expressed as a percentage of the non-fluctuating velocity. This is expressed as:

$$T_u = \frac{\sqrt{u^2}}{U}$$
 [1]

Roach developed correlations for predicting turbulence intensity generated by passive grids (Roach 82-92). For a square mesh of square grids this correlation is Reynolds number independent. This correlation is given as:

$$T_{u} = C \left(\frac{x}{d}\right)^{-5/7}$$
 [2]

is a correlation constant which for square mesh of square bars is 1.13, d is the ance or bar diameter, and x is the distance downstream of the grid (Roach 84-ion allows for the prediction of turbulence intensity or for this study the distance are a desired turbulence intensity.

rarameters of interest are the micro and macro or integral length scales. The is the dissipation scale. It may be considered a measure of the average length the are primarily the method in which turbulence energy is dissipated (Roach 85). described by the equation:

$$\frac{1}{\lambda_u} = \frac{-1}{2U} \left\{ \frac{\partial^2 R(T)}{\partial T^2} \right\}_{T=0}$$
 (Roach, 85) [3]

(T) represents the autocorrelation function, a correlation of each point to all time. The second derivative of this correlation is used to determine the conse of this function.

ral length or macro scale may be considered to be a measure of the largest eddy ent field (Roach, 85). This is found by determining the area under the urve as it goes from 1 to 0. This area is determined by integrating this curve as ne macro length scale is defined as:

$$\Lambda_{u} = U \int_{0}^{\infty} R(T) dT$$
 [4]

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The heat transfer coefficients are of direct interest to this investigation. For this study the blade will be heated by resistance heating to provide a surface temperature higher than the surrounding air temperature. This temperature difference will in turn drive convective heat transfer to occur. The amount of resistance heating can be calculated as follows:

$$q' = \frac{I^2 R_{35.7}^{"}}{w^2}$$
 [6]

In this equation q" is the heat transfer from the blade, I is the current applied to the blade, R"_{35.7} is the resistance per square of the gold, and w is the width of the gold layer. This represents the total heat transfer from the blade. In order to determine the convective heat transfer which is the target of this investigation the conductive and radiative nodes of heat transfer must be accounted for. Since, the construction of the turbine blade is designed to minimize conduction by using a highly insulative closed-cell foam core, that mode is assumed to be negligible. The radiative mode is calculated by the following equation:

$$q_R^* = \varepsilon \sigma \left(T_{LC}^4 - T_{\infty}^4 \right) \tag{7}$$

where T is the absolute temperature in Kelvin for both cases. By subtracting these modes from the total heat transfer the conductive heat transfer can be determined.

$$q_c^* = q^* - \varepsilon \sigma \left(T_{LC}^4 - T_{\infty}^4 \right) - q_L^*$$
 [8]

Once the conductive heat flux is determined the conductive heat transfer coefficient can be determined as follows.

$$h = \frac{q_c^*}{\left(T_{LC} - T_{\infty}\right)}$$
 [9]

The next step in the process is to non-dimensionalize this coefficient. This can be reduced to either the Stanton number or Nusselt number for reporting. These two are determined as follows:

$$St = \frac{h}{\rho V_{\infty} c_p}$$
 [10]

$$Nu = \frac{hB_x}{k_{air}}$$
 [11]

The Nusselt number can be divided further by the square root of Reynolds number to remove any Reynolds number dependency in laminar flow regions.

Experimental Background

This investigation builds on work previously performed at both the United States Air

Force Academy and the University of California at Davis. This work includes investigating the effects of turbulence intensity on heat transfer as well as mapping the turbulence generated by grids. Baughn et al investigated the effect of turbulence intensity in a simultaneous study at UCDavis and USAFA. Their results can be summarized as the turbulence intensity is increased from 1% to 10% the heat transfer level increases, the suction side boundary layer transition moves upstream and the spanwise variation on the pressure side disappears (Baughn, et al 12). Baughn et al also note that these results compare favorably to rotating tests performed. This helps address the concern that cascade tunnels do not address rotational effects (Baughn, et al 12).

The study of turbulent flow quality produced by grids conducted last semester by Duncan and Peterson confirmed that the work done by Roach held true for the Cascade Wind Tunnel.

The results of that investigation concluded that the correlation given by Roach worked well for passive generation such as will be used in this investigation. It also noted that the use of a grid

passive generation such as will be used in this investigation. It also noted that the use of a grid perpendicular to the flow instead of parallel to the cascade had negligible effects on the turbulence generated (Duncan and Peterson 18-19). That research used a round bar generation grid and thus the correlation coefficient is Reynolds number dependent as shown in Roach (Roach 86). This research tests a square mesh of square bars which Roach found to be Reynolds number independent in the range of interest (Roach 87).

With the effect of turbulence intensity investigated by Baughn et al and the effectiveness of generating turbulence by grids established by Roach and confirmed for the USAFA tunnel by Duncan and Peterson, this investigation establishes a relationship between length scales and heat transfer. By using a turbulence intensity level of 10%, the results of this investigation can be compared to Baughn et al, to analyze the effects of length scales. The objective of this project is to determine if a relationship exists between turbulent heat transfer and micro and macro length scales. Heat transfer is tested at 10% turbulence intensity. Additionally, a clean tunnel test is performed for comparison. Length scale comparisons will allow for a better understanding of turbulent heat transfer. Length scale investigations will provide data for updating turbulent CFD codes to include length scale effects. The investigation also looks at passive turbulence generation, comparing the results to the correlations reported by Roach (Roach 82-92).

Experimental Methods

Experiment Setup

The experiment takes place in the USAF Academy Aeronautics Lab Cascade Wind

Tunnel. The Cascade Wind Tunnel is designed to place a linear cascade of turbine blades in a

flow which will simulate the flow over the normally rotating turbine blades. The turbines in the

turbine around the 3rd or 4th stage with Reynolds numbers around 80,000. The tunnel is a closed loop system with a heat exchanger to provide for temperature control. In the test section are seven test blades with the end walls simulating two more blade surfaces. This setup is shown in Figure 1. The blades in the test section very closely match the Langston geometry. This geometry can be seen in Langston et al, 1977 (Langston 23).

The test section has a test probe which traverses parallel to the blade plane. This probe contains a hot film sensor, pitot-static probe and a wedge probe, which is not used in this study. The pitot-static probe is connected to a pressure transducer which outputs to a torr meter as well as the IFA 100. The hot film anemometer outputs to the IFA 100 and HP 3852A. Additionally a K type thermocouple outputs to the IFA 100 allowing for temperature measurements. Data from the IFA 100 and HP 3852A are input to the P90 computer which is controlled by TV3 software, which was written in the Aero Department. This setup is designed to collect the flow data. This can be seen in Figure 2 and Figure 3.

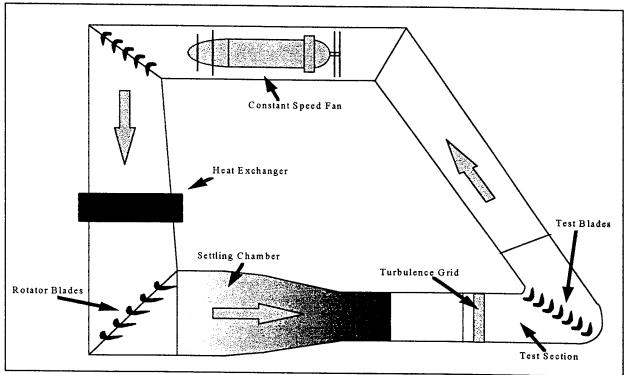


Figure 2: Cascade Wind Tunnel Schematic

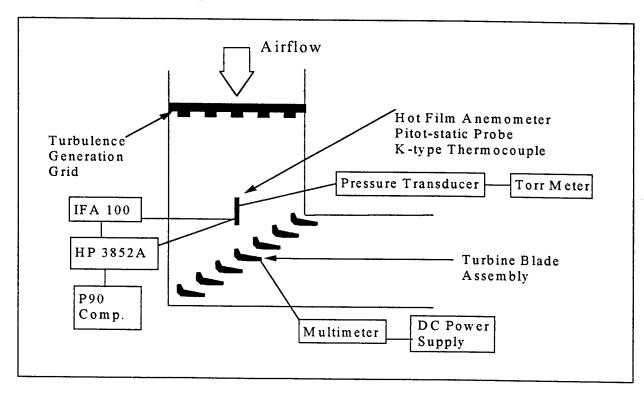


Figure 3: Test Section Diagram

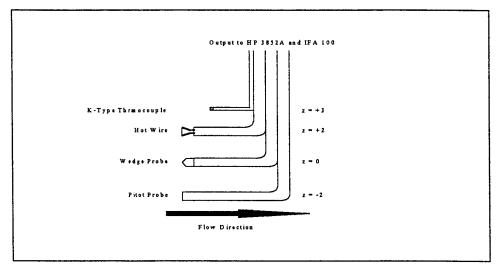


Figure 4: Traversable Test Probe

The setup for collecting heat transfer data centers on the test blade. The test blade is located in the center of the cascade. It is made of closed cell polystyrene, which is covered with Super 77 aerosol adhesive and a thin gold film with a resistance per square of 2.512 ohm/sq. This gold film is then covered with black paint and sprayed with 35W1 liquid crystals. The liquid crystals are active in the 35-36° Celsius range, with the yellow band indicating 35.7° Celsius. The gold acts as a conductor and provides resistance heating to the blade. Two electrodes at the trailing edge are connected to a DC power supply providing up to 5 amps. A multimeter is placed in series with the power supply and the blade to provide current readings. The entire test blade setup can be seen in Figure 4. This test blade allows for surface temperature a current readings to be taken which then allow heat transfer calculations to be performed.

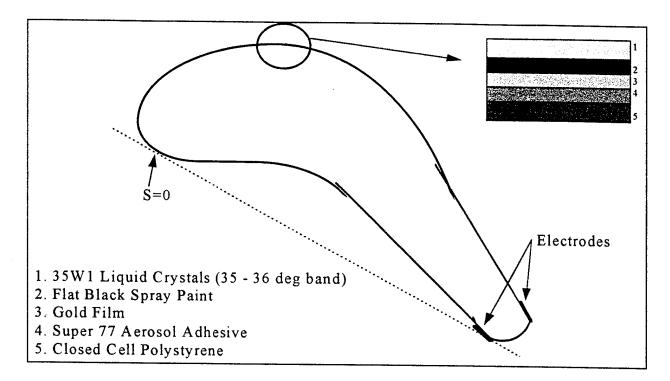


Figure 5: Test Blade Setup

The turbulence grids are constructed of square wood pieces which are nailed together in a square grid lattice. Two turbulence grids were constructed for this procedure, one grid of 1/2 in. diameter bars and one grid of 2 3/8 in. diameter bars.

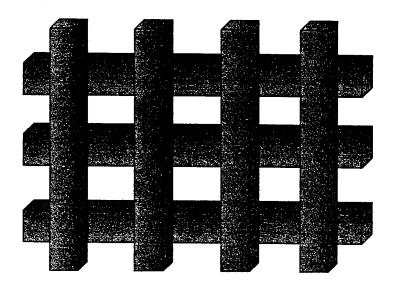


Figure 6: Grid of 2 3/8 in Diameter Bars

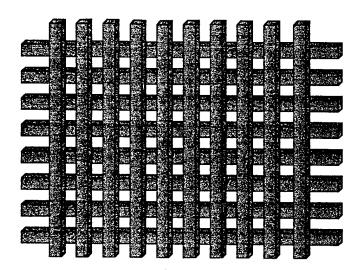


Figure 7: Grid of 1/2 in Diameter Bars

The full list of equipment used in this investigation is as follows:

Table of Equipment Cascade Wind Tunnel, U.S. Air Force Academy Aeronautics Lab 1/2 in. and 2 in. square mesh of square bars turbulence generation grids Pitot-static tube Pressure transducer Torr pressure gage, SN 54258-1 Hot film anemometer, TSI Model 1210-20, SN 925113 K type Thermocouples IFA 100 Intelligent Flow Analyzer HP 3852A data acquisition/control unit IBM compatible Pentium 90 computer TV3 Software SPSS Statistical Software Microsoft Excel 4.0 Software Styrofoam turbine blade assembly (See Figure 4)

HP 6434B DC power supply Fluke 27/FM Multimeter

Table 1: List of Equipment

The instrument calibration for this experiment was already performed by laboratory personnel annually. The hot wire anemometer calibration is a fourth order curve fit which had been previously determined. The thermocouples, pressure transducers, thermometers and

barometers were calibrated by USAFA PMEL. The liquid crystals on the blade were also calibrated previously.

Experimental Procedure

To obtain turbulent heat transfer and flow data use the following procedure:

- a: Using a clean tunnel configuration place the hot wire portion of the test probe at the center line position in front of the test blade. Ensure that the hot wire is directly into the flow. Raise the hot wire to a vertical position in the center of the gold section, (z = 9 in.). With the probe located in this position take zero velocity data for the hot wire, pitot-static probe and temperature. Note the atmospheric pressure.
- b. Turn on the air allowing the tunnel to spool up to steady state (approximately 9 m/s). Then setting the TV3 software to take continuous data for 2.5 seconds at 6000 Hz on the hot wire, take hot wire, pitot-static and temperature data. Ensure that the temperature remains close to the zero velocity data. This data will be used to determine the integral length scale and turbulence intensity. In the same position set the TV3 to take continuous data at 15000 Hz for 1 second on the hot wire and take the same types of data. This data will be used to analyze the micro length scales and turbulence intensity. Note the turbulence intensity should be the same.
- c. Using the cooling flow, allow the tunnel to reach a steady state temperature of 25 Celsius. adjust the cooling flow to ensure that this temperature is maintained.
- d. When the tunnel has reached steady state carefully apply current from the power source to the test blade until the liquid crystals become active. When the liquid crystals are active note the position of the yellow band on both the suction and pressure sides as well as the current being applied. By gently adjusting the current, the yellow band can be moved over the entire blade

noting the position and current at each position on the blade. Note any indications of flow phenomenon such as transition and Taylor-Görtler vortices. Use care not to overload the blade as it could cause the foam core to melt and ruin the test blade.

- e. Place the 2 3/8 in grid in the wind tunnel at the previously calculated distance for 10% turbulence intensity (approximately 72 in.). This is as far back from the blade as the test section allows. Take zero velocity and turbulent flow data as described above. Using SPSS analyze the continuous data to determine the turbulence intensity.
- f. Move the data probe out of the flow and adjust the position of the grid forward to account for the distance between the probe and the turbine blade so that the blade itself is now seeing 10% turbulence intensity.
- g. Run the tunnel up to steady state as describe for the clean tunnel case and take heat transfer data using the same procedure described in the clean tunnel case.
- h. Replace the 2 3/8 in. turbulence generation grid with the 1/2 in. turbulence generation grid and repeat the same procedure used for the 2 3/8 in. grid. Take turbulence data and move the grid as necessary to match the large grid turbulence intensity.
- Reduce all data for all flows and heat transfer to determine any possible relationship between length scales and heat transfer.

Uncertainty Analysis

The uncertainty analysis of the heat transfer data was performed using the Kline-McClintock method. The equations of interest here are as follows:

$$q'' = \frac{I^2 R_{35.7}^{"}}{w^2}$$
 [12]

$$q_c^* = q^* - \varepsilon \sigma \left(T_{LC}^4 - T_{\infty}^4 \right) - q_L^*$$
 [13]

$$h = \frac{q_c}{\left(T_{LC} - T_{\infty}\right)}$$
 [14]

The equations were then placed in an Excel 4.0 spreadsheet along with the uncertainty equations in order to determine the expected uncertainty. Since our current ranges from 2.0 amps to 5.0 amps, the calculated uncertainty as per the spreadsheet in h ranges from 9.01% at 2.0 amps to 5.64% at 5.0 amps. This is well within acceptable parameters for this type of experiment. This spreadsheet is shown below in Figure 5.

					%Uncertainty	q" Uncertainty	%q" Uncertainty	qc" Uncertainty	%qc" Uncertainty	h Uncertainty	%h Uncertainty
Current			0.0050	Amps	0.10	3.01	0.20	301	0.21	0.28	0.21
Resistance	251	+/-	0.1256	Ohme/sq	5.00	75.31	5.00	75.31	5.20	7.04	
Film Width			0.0002		0.10	-293	-0.19	-293	-0.20		
Tinf			0.1500		0.60	Not Used	Not Used	0.77	0.05	1.97	
TLC			0.1500		0.42	Not Used	Not Used	-0.85	-0.06	-1.98	
Emmisivity			0.1500		17.65	Not Used	Not Used	-10.16	-0.70		
Conduction	0.00	+/-	0.0000	Wm ²	0.00	Not Used	Not Used	0.00	0.00		
S-B Const	5.67E-08	W	(m/2"K"	4)							
q " = -	1506.219	+/-	75.428	Wm²2	5.01	%Uncertainty					
qc" =	1448.624	+/-	76.118	Wmt2		%Uncertainty					
h= =	135,385	+/-	7.641	W(m²K)		%Uncertainty					

Figure 8: Uncertainty Analysis

Results and Discussion

The results and discussion will be presented in three separate sections for each of the cascade tunnel configurations (clean, ½ in. grid, 2 3/8 in grid). After each configuration is presented and discussed and overall evaluation and discussion of the experiment will be presented.

"Clean" Tunnel Configuration (No Turbulence Grid)

The turbulence data for all runs was taken at the center of the tunnel with the hot film anemometer aligned with the center of the test blade. All data is taken at this point over time. The key flow information for the clean tunnel run is tabulated below:

Flow Information	***************************************
Atmospheric Pressure	11.183 psia
Hot Film Zero Voltage	-5.351 V
Torr Meter Delta P	0.280 Torr
Mean Velocity	8.807 m/s @ 6000 Hz
	8.799 m/s @ 15000 Hz
Standard Deviation	0.044 m/s @ 6000 Hz
	0.040 m/s @ 15000 Hz
Turbulence Intensity	0.50% @ 6000 Hz
Micro-Length Scale	Infinite
Integral Length Scale	0.00543 m @ 15000 Hz
	0.214 in @ 15000 Hz

Table 2: Clean Tunnel Flow Information

With this turbulence data taken, the heat transfer data was then taken and reduced. This data provided a baseline from which the effects of the turbulence could later be determined. This data was plotted against data provided by Capt Butler which was taken last semester. The data agreed nicely with last semester's data in all points but the stagnation point, which appeared to high on our run. Since this is an important point and there was significant fluctuation in temperature during the test run, it was decided to retest this point at several temperatures. This would determine whether there was a temperature dependency that was not removed by the data reduction or whether this was just an errant point. The results of the stagnation collection are shown below. The stagnation data points were relatively flat when retaken vs. temperature and all variation was within our experimental error. This supports the previous expectations. The average of these points was then entered as the stagnation heat transfer.

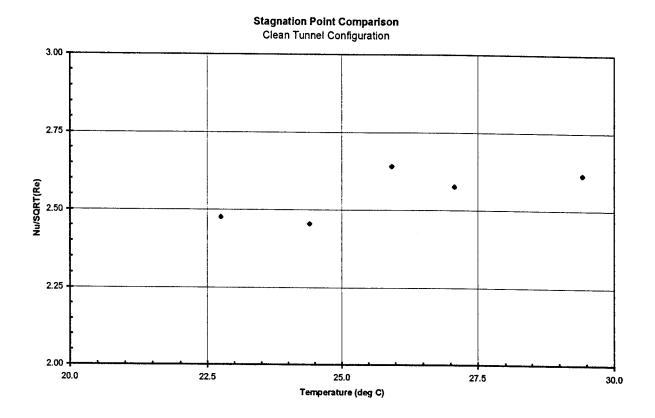


Figure 9: Stagnation Point Comparison

With the new stagnation point all the data for the clean tunnel lined up within the expected error and provided confidence in the data gathering technique. Some significant observations included an area on both the pressure side and the suction side where the foil had bubbled at the midspan of the blade. This bubbling caused flow irregularities such as separation and premature transition of the boundary layer. To account for this the data was taken from the top portion of the blade which was unaffected by these irregularities. While this does not necessarily invalidate the data it could be a source of error that later studies must consider.

Also of interest was the formation of spanwise variations on the pressure side of the blade that could represent Taylor-Görtler vortices. This is represented on the plot by the range of data points for a given Nusselt number.

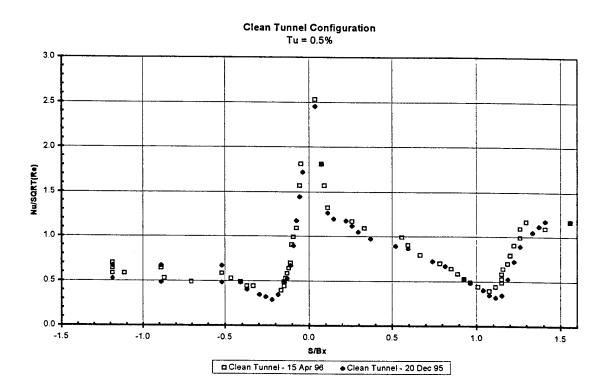


Figure 10: Clean Tunnel Comparison

2 3/8 in Grid Configuration

With the 2 3/8 in diameter bar turbulence grid in the test section, the turbulent flow data was reduced to give the following flow information.

Flow Information	
Atmospheric Pressure	11.216 psia
Hot Film Zero Voltage	-5.36 V
Torr Meter Delta P	0.276 Torr
Mean Velocity	8.882 m/s @ 6000 Hz
	8.658 m/s @ 15000 Hz
Standard Deviation	0.831 m/s @ 6000 Hz
	0.881 m/s @ 15000 Hz
Turbulence Intensity	10.18% @ 6000 Hz
Micro-Length Scale	0.07053 m @ 6000 Hz
7	2.777 in @ 6000 Hz
Integral Length Scale	0.00837 m @ 15000 Hz
***************************************	0.330 in @ 15000 Hz

Table 3: 2 3/8 in Grid Flow Information

This flow represents high turbulence intensity with large length scales and as expected affected the turbulence intensity by moving the transition point forward on the blade, and increasing heat transfer. The transition points moved forward 0.1875 in on the pressure side and 0.625 in on the suction side. At the stagnation point the heat transfer was increased by 6.25%. The spanwise distribution on the pressure side disappeared and post-transition heat transfer on the pressure side was dramatically increased by as much as 52 %. Elsewhere, increases in heat transfer were more moderate, on the order of the stagnation point increases. This distribution is shown in Figure 11.

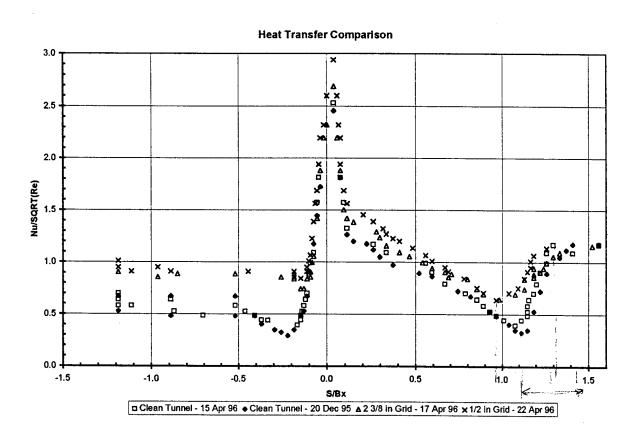


Figure 11: Heat Transfer Comparison

1/2 in Grid Configuration

With the 1/2 in turbulence grid in place the flow data was reduced to give the following flow information:

Flow Information	
Atmospheric Pressure	11.216 psia
Hot Film Zero Voltage	-5.34 V
Torr Meter Delta P	0301 Torr
Mean Velocity	9.347 m/s @ 6000 Hz
	8.998 m/s @ 15000 Hz
Standard Deviation	0.949 m/s @ 6000 Hz
	0.894 m/s @ 15000 Hz
Turbulence Intensity	10.15% @ 6000 Hz
Micro-Length Scale	0.01320 m @ 6000 Hz
	0.520 in @ 6000 Hz
Integral Length Scale	0.00490 m @ 15000 Hz
	0.193 in @ 15000 Hz

Table 4: 1/2 in Grid Flow Information

The heat transfer data for this run, also plotted in Figure 11, shows the same trends as the 2 3/8 in grid. This is expected due to the nearly identical turbulence intensity. With this grid, however, the length scales are much smaller. The 1/2 in grid produced higher heat transfer rates than the 2 3/8 in grid until transition, which occurred at the same points as with the large grid. After transition the heat transfer characteristics were the same as the large grid characteristics. At stagnation the heat transfer was 16.2% higher than the clean tunnel and 9.4% higher than that produced by the large grid. Similarly the other data points before transition were higher than both the large grid and clean tunnel. Since this difference is greater than our uncertainty this represents a significant finding on the affect of length scales on heat transfer. The differences between the two grids are better highlighted in Figure 12 where just those two data sets are plotted. Additionally, lines are added to highlight the differences.

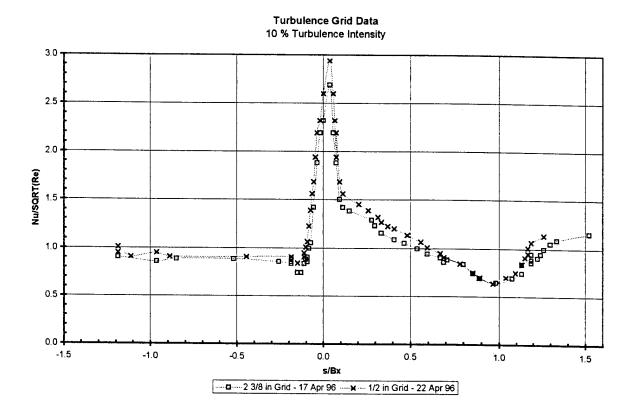


Figure 12: Grid Comparison

Overall Discussion

The heat transfer characteristics of the turbine blade responded to turbulence intensity n the same manner as had already been shown by previous work such as Baughn et al. The effects of length scales appear to be primarily in the region of pre-transition heat transfer. The transition points are unaffected by the length scales, but a shorter length scale appears to cause a significant increase in heat transfer at the same turbulence intensity. This increase lasts until boundary layer transition where the free-stream turbulence become insignificant compared to the turbulent boundary layer.

Conclusions

The effects of operating a turbine blade under high turbulence intensity, such as the 10.15% to 10.18% used in this investigation are an increase in heat transfer over the entire blade, the boundary layer transition points move forward on both blade surfaces, and the disappearance of spanwise heat transfer variations on the pressure side. In addition to these documented effects, it appears that smaller length scales at the same turbulence intensity produces higher heat transfer than that produced by longer length scales. In this test the stagnation temperature produced by the 1/2 in grid was 9% higher than that produced by the 2 3/8 in grid. The length scales appear to have no effect on transition location and the effects of differing length scales disappear after boundary layer transition. This is most likely due to the relative insignificance of the free-stream turbulence compared to the boundary layer turbulence. This investigation shows that length scales therefore have a significant impact on pre-transition heat transfer in turbine blades.

Recommendations

The investigation of the effects of length scales on heat transfer should be continued.

Tests should be conducted at different turbulence intensity levels and with more gradations of length scale in order to try an quantify the effect of length scales on the turbine heat transfer.

Improvements on this experiment would be to include a better blade so that data can be taken over the entire blade. Also the people taking data could be more familiar with liquid crystals to allow for better readings of the crystals. Where possible have the same points measured by the same person in order to eliminate error introduced by having different people view the crystals. A greater understanding of length scales is essential to understanding turbulent heat transfer, especially since this investigation has shown that they have a significant impact on heat transfer.

Acknowledgments

We would like to acknowledge our sponsors, LtCol Barlow, LtCol Van Treuren, and Capt Butler, for their patient instruction and support for this project. Also we give thanks to SSgt Evans for providing facility support.

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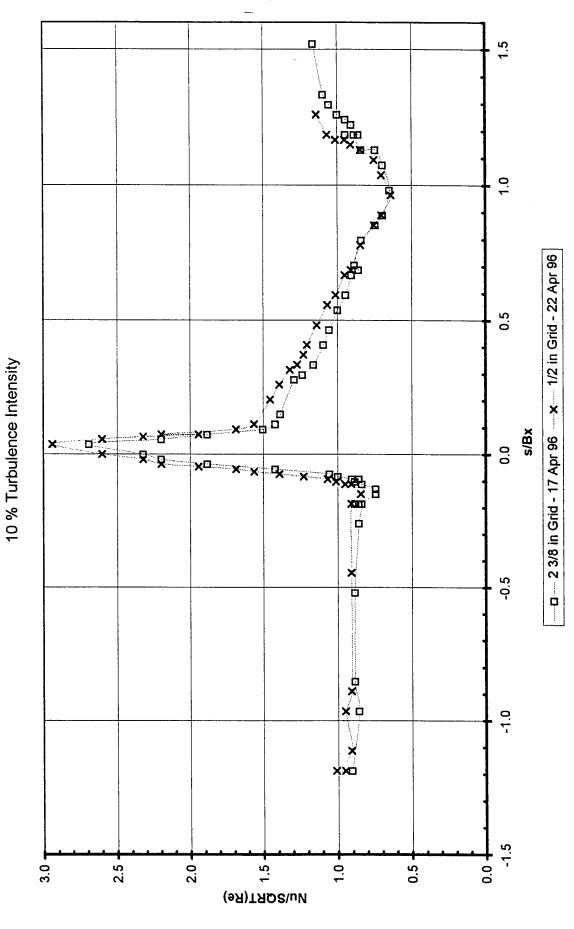
 NASA Technical Memorandum, No. 86884

Appendix A: Heat Transfer Graphs

30.0 27.5 Stagnation Point Comparison Clean Tunnel Configuration Temperature (deg C) 25.0 22.5 20.0 2.00 + 2.75 М_U/SQRT(Re) ? % 2.25

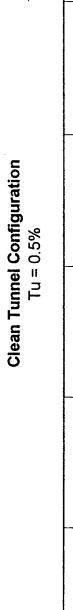
<u> </u>	44.044			Db	0.0127	ke/m^2			
P inf =	11.311			Rho = Mu =	0.9137 1.8337E-05				
T inf =	24.400			k air =	0.0261				
T Ic =	35.700			V inf =	9.215				
Delta P =	0.291			Bx =	6.740				
L=	18.531			Re =	78610.3				
W =	8.039					W/m^2			
R" =		Ohms/SQ		Cond. =		W/m^2K^4			
Emmis. =	0.850			S-B Con. =	5.0702-06	VV/III**2N**4			
	10 5	- 6	Ol T	20 D (NE				
Clean Tunn	iei Contigur	ation -	Clean Tunn	nel - 20 Dec 9	5				
- (i=)	-/D.	Current (Amne)	Tinf (C)	a" (\Allm\2)	qc" (W/m^2)	h (M/m^2K)	St	Nu	Nu/SQRT(Re)
s (in) -8.000	s/Bx -1.1869	Current (Amps) 2.29	24.4	315.951	255.304		0.002665	148.152	0.5284
			24.4	385.646	325.000				0.6727
-8.000	-1.1869 -0.8902		24.4	294.261	233.615				0.4835
-6.000				385.646	325.000		0.002400		0.6727
-6.000 -3.500	-0.8902 -0.5193		24.4 24.4	294.261	233.615				
			24.4	385.646	325.000				
-3.500			24.4	294.261	233.615				
-2.750 -2.500	-0.4080 -0.3709			255.672	195.025				0.4036
	-0.3709			229.096	168.450				0.3486
-2.000 -1.750	-0.2967 -0.2596			217.498	156.852				
-1.750	-0.2396			201.767	141.121				
-1.250				229.096	168.450				
-1.250			24.4	294.261	233.615				
-0.875				315.951	255.304				
-0.875				385.646	325.000				
-0.730				492.811	432.164				
-0.500				628.569	567.923				
-0.375				759.285	698.639				
-0.250				893.037	832.391				
0.250				1247.300		105.014			
0.500				935.278					
0.750				672.111	611.465				
1.000				640.300					
1.500				628.569	567.923				
1.750				601.620					
2.000				567.839					
2.500				531.448					
3.500			+						
4.000				479.122					
5.000				410.421	349.774				0.7239
5.500							0.003392		
6.250							0.002665		
6.500					+				
7.000									
7.250									
7.500									
7.750									
8.000									
8.250									
8.500									
9.000									
9.250									
9.500									
10.500							0.005928		

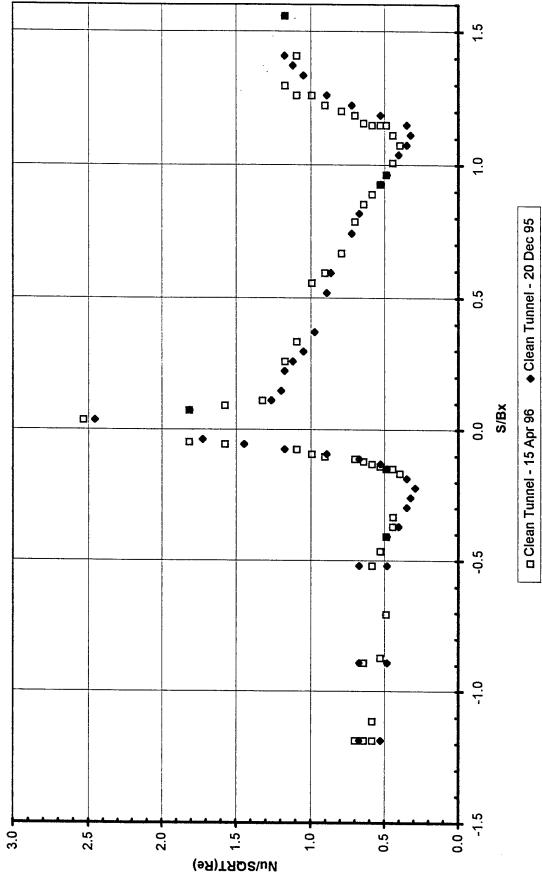
Turbulence Grid Data



P inf =	11.216	nsi		Rho =	0.9039	kg/m^3			
T inf =	25.100			Mu =	1.837E-05				
T ic =	35.700			k air =	0.0262	W/mK			
Delta P =	0.276			V inf =	9.023				
L=	18.531			Bx =	6.740	in			
W =	8.039		:	Re =	76007.9				
R" =	2.512	Ohms/SQ		Cond. =		W/m^2			
Emmis. =	0.850		5	S-B Con. =	5.670E-08	W/m^2K^4			
2 3/8 in. Gr	d at 72 3/4	in	2 3/8 in Grid	- 17 Apr 96					
									11 (00 D T/D)
s (in)	s/Bx	Current (Amps)	T inf (C) q		qc" (W/m^2)	h (W/m^2K)	St		Nu/SQRT(Re)
-8.000	-1.1869	2.79	25.0	467.303	409.504		0.004642		0.9051
-6.500	-0.9644	2.73		447.385	389.382		0.004398		0.8574
-5.750	-0.8531	2.70		439.213			0.004550		
-3.500	-0.5193	2.70		439.213	383.817		0.004550		0.8871
-1.750	-0.2596		24.9	447.385					0.8574
-1.250	-0.1855		25.4	419.910			0.004302		
-1.250	-0.1855			439.213	383.817		0.004550	244.579	
-1.250	-0.1855			447.385			0.004398	236.387	0.8574
-1.000	-0.1484			382.604			0.003814	205.022 205.022	0.7437 0.7437
-0.875	-0.1298			382.604	326.439		0.003814		
-0.750				419.910	364.308		0.004302 0.004550		
-0.688				439.213			0.004398		
-0.625				447.385			0.004596		
-0.625				467.303			0.004042		
-0.563				510.193			0.005405		
-0.500				535.033			0.007273		
-0.375			25.1	688.306					
-0.250		3.85		893.037			0.00303		
-0.125		4.14	25.2	1032.640			0.011202		
0.000				1085.684 1297.132					
0.250				1032.640			0.011262		
0.375				893.037					
0.500				725.449					
0.625				688.306					
0.750				674.125					
1.000				636.378					
2.000									
2.25				582.73					
2.75									
3.12				535.03		3 44.38	9 0.00540	5 290.498	
3.62								8 275.08	
4.00									
4.50									
4.62				447.38					
4.75									
5.37					0 364.30				
5.75									
6.00				361.64					
6.62							4 0.00332	1 178.48	
7.25			5 25.3	361.64			0.00356		
7.62			25.3				8 0.00381		
7.62		13 2.6					0.00430		
8.00		39 2.7					3 0.00455		
8.00									
8.00			35 24.9						
8.25							0.00464		
8.37									
8.50							35 0.00511		
8.79							0.00540		
9.00		53 3.0					0.00560		
10.2			11 24.9	582.73	524.83	31 48.7	76 0.00593	39 319.20	1.1578

P inf =	11.396	nei		Rho =	0.0170	kg/m^3			
T inf ≃	25.250			Mu =					
T Ic =	35.700			k air =	0.0262				
Delta P =	0.277			V inf =	8.970				
L =	18.531			Bx =	6.740				
W =	8.039			Re =	76705.1				
R" =		Ohms/SQ		Cond. =					
Emmis. =	0.850	Offinistod		S-B Con. =		W/m^2 W/m^2K^4			
Limit, -	0.000			3-B COII	3.070E-06	VV/III-2K-4	ļ		
Clean Tunne	Configura	tion	Cloop Tupe	nel - 15 Apr 96	=			ļ	
Olean Tunne	Comguia	BO11 -	Clean Tuni	lei - 15 Apr se	7				
s (in)	s/Bx	Current (Amps)	Tinf (C)	q" (W/m^2)	ao" (\A//m\A2)	h (14/1-4016)	CL	NI	11. (0.0 0 77.0
-8.000	-1.1869	2.36			qc" (W/m^2)			Nu 464.007	Nu/SQRT(Re
-8.000	-1.1869	2.45	24.5 24.6		275.575		0.002975		0.5827
-8.000	-1.1869	2.53			301.809				0.6399
-7.500	-1.1128	2.36	24.7		326.523		0.003580		
-6.000	-0.8902		24.5	335.562	275.575		0.002975		
-5.000 -5.875		2.45	24.6		301.809				
	-0.8717	2.27							
-4.750 3.500	-0.7047	2.20	24.5				0.002490	135.041	0.4876
-3.500	-0.5193	2.36							0.5827
-3.125	-0.4636	2.27	24.5		250.317	22.350			0.5279
-2.750	-0.4080	2.20	24.5	291.604		20.644	0.002490		0.4876
-2.500	-0.3709	2.12	24.5		210.542	18.765			0.4432
-2.250	-0.3338	2.02	25.5						
-1.125	-0.1669	2.02	24.6						0.3948
-1.000	-0.1484	2.12	24.5				0.002263		0.4432
-1.000	-0.1484	2.20	24.5		231.414		0.002490		0.4876
-0.938	-0.1391	2.27					0.002695		0.5279
-0.875	-0.1298	2.36	24.5		275.575				0.5827
-0.813	-0.1205	2.45	24.6	361.643	301.809		0.003267	177.226	
-0.750	-0.1113	2.53		385.646	326.523	29.684	0.003580	194.179	
-0.688	-0.1020	2.76		458.951	402.378	38.322	0.004622	250.684	0.9051
-0.625	-0.0927	2.86	25.3		436.749	41.995	0.005065	274.713	0.9919
-0.500	-0.0742	2.97			475.898	46.204	0.005572	302.245	1.0913
-0.375	-0.0556	3.42				66.501	0.008020	435.021	1.5707
-0.313	-0.0464	3.64	26.0	798.272	745.701	76.718	0.009253		1.8120
0.250	0.0371	4.36			1092.477	111.820	0.013486		2.5312
0.500	0.0742	3.64	26.0	798.272	745.701	76.718			
0.625	0.0927	3.42		704.694	651.711	66.501			1.5707
0.750	0.1113		25.8	605.434	552.142	55.998	0.006754	366.316	1.3226
1.750	0.2596	3.05	25.5	560.464	505.426	49.552			
2.250	0.3338	2.97			475.898	46.204	0.005572	302.245	1.0913
3.750	0.5564			492.811	436.749	41.995	0.005065	274.713	
4.000	0.5935	2.76		458.951	402.378		0.004622	250.684	
4.500	0.6677	2.63	25.0	416.735			0.004048	219.564	
5.300	0.7864			385.646	326.523		0.003580		
5.750		2.45							+
6.000	0.8902								
6.250	0.9273	2.27							
6.500	0.9644	2.20							
6.800	1.0089							122.751	
7.250	1.0757	2.02					0.002016		
7.500	1.1128								0.4432
7.750									0.4876
7.750									
7.750	1.1499								
7.800	1.1573								
8.000	1.1869								
8.100	1.2018						0.003080		
8.250								219.564	
8.500		2.86							
8.500		2.97					0.005065		
8.750		3.05						302.245	
9.500		 					0.005976		1.1704
						·	0.005572		
10.500	1.5579	3.05	25.5	560.464	505.426	49.552	0.005976	324.145	1.1704





P inf =				Rho =		.kg/m^3			
T inf =				Mu =	1.8384E-05	N s/m^2			
T Ic =	35.700			k air =	0.0262				
Delta P =	0.301			V inf =	9.344	m/s			
L≠	18.531	in		Bx =	6.740	in			
W =	8.039	in		Re =	79855.6			-	
R" =	2.512	Ohms/SQ		Cond. =		W/m^2			
Emmis. =	0.850			S-B Con. =		W/m^2K^4			
									
1/2 in Grid	at 15 in -		1/2 in Grid	- 22 Apr 96					
s (in)	s/Bx	Current (Amps)	Tinf(C)	a" (W/m^2)	gc" (W/m^2)	h (W/m^2K)	St	Nu	Nu/SQRT(Re)
-8.000		2.81			420.539		0.004760		
-8.000									0.9512
-7.500					403.606		0.003000		1.0111
-6.500					420.539		0.004360		0.9102
-6.000		2.76		<u> </u>	403.606		0.004760		0.9512
-3.000		2.76							0.9102
-1.250					403.606		0.004555		0.9102
-1.000					403.606		0.004555		0.9102
-0.750					374.213		0.004228		0.8448
					403.606				0.9102
-0.750					420.539				0.9512
-0.688		2.89			447.910		0.005060		1.0111
-0.625		2.96			472.582		0.005339		
-0.563		3.16			546.070		0.006139		1.2267
-0.500		3.35					0.006955		1.3898
-0.438					691.264		0.007817		1.5620
-0.375					751.467				1.6865
-0.313					878.858				1.9423
-0.250		4.15			981.828		0.010985		2.1950
-0.125							0.011621		2.3220
0.000					1164.180	112.373			2.6002
0.250		4.78			1320.577				2.9410
0.375					1164.180	112.373	0.013013	734.781	2.6002
0.438					1037.616	100.350	0.011621	656.165	2.3220
0.500			25.2		878.858			548.870	
0.500					981.828	94.863	0.010985	620.287	2.1950
0.625			25.4	807.068	751.467	72.887	0.008440	476.594	1.6865
0.750	0.1113	3.52	25.5	746.506	691.264	67.506		441.409	1.5620
1.375	0.2040	3.43	25.3	708.821	652.810	62.831			
1.750	0.2596	3.35	25.4		620.438	60.062			1.3898
2.125	0.3153	3.26					0.006609		
2.250		3.21					0.006360		
2.500				601.620			0.006139		
2.750							0.006021		
3.250	+						0.005688		
3.750							0.005339		
4.000							0.005060		
4.500							0.004760		
4.625							0.004555		
5.250	·						0.004333		
5.750							0.004228		
6.000							0.003772		
6.500							0.003324	·	
7.000							0.003198		
7.375							0.003524		
7.625	·								
7.750		+					0.004228		
7.875	+						0.004555		
7.875	 						0.004760		
		+					0.005060		
8.000							0.005339		+
8.500	1.2611	3.06	25.4	564.145	508.339	49.115	0.005688	321.152	1.1365

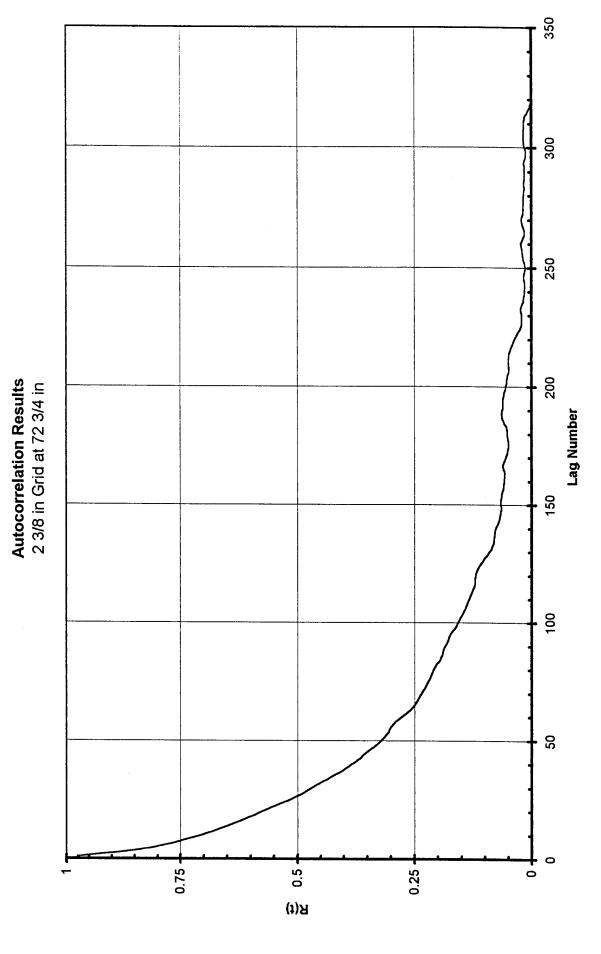
Stagnation Point - 01 May 96

P inf =	11.280	psi		Rho =	0.9093	kg/m^3			
T inf =	25.000	С		Mu =	1.8365E-05	N s/m^2			
T Ic =	35.700	С		k air =	0.0262	W/mK			
Delta P =	0.323	torr		V inf =	9.732	m/s			
L=	18.531	in		Bx =	6.740	in			
W =	8.039	in		Re=	82494.8				
R" =	2.512	Ohms/SQ		Cond. =	0	W/m^2			
Emmis. =	0.850			S-B Con. =	5.670E-08	W/m^2K^4			
Clean Tunr	nel Stagnatio	on Points -		01 May 96,	15 Apr 96, & 2	0 Dec 96			
s (in)	s/Bx	Current (Amps)	T inf (C)	q" (W/m^2)	qc" (W/m^2)	h (W/m^2K)	St	Nu	Nu/SQRT(Re)
0.250	0.0371	4.12	27.1	1022.687	975.762	113.066	0.012687	740.154	
0.250	0.0371	3.54	29.4	755.013	720.473	114.725	0.012874	751.012	
0.250	0.0371	4.95	22.8	1476.245	1407.302	108.672	0.012194	711.388	2.4768
0.250	0.0371	4.36	25.9	1145.305	1092.477	111.820	0.013486	731.474	
0.250	0.0371	4.55	24.4	1247.300	1186.654	105.014	0.012385	688,608	2,4560

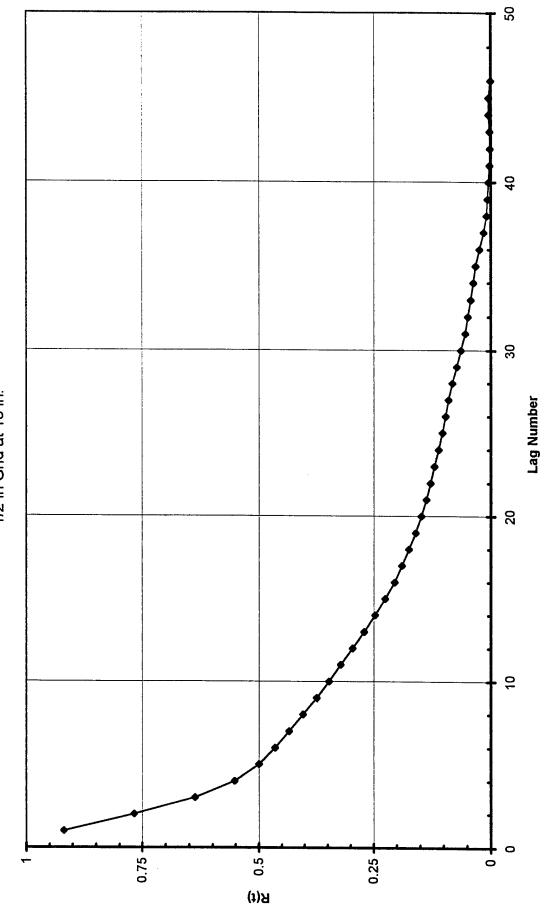
Appendix B: Heat Transfer Data Sheets

1.5 □ Clean Tunnel - 15 Apr 96 ◆ Clean Tunnel - 20 Dec 95 ▲ 2 3/8 in Grid - 17 Apr 96 × 1/2 in Grid - 22 Apr 96 0. Heat Transfer Comparison × & S/Bx **∀**×□◆ × 0.0 ð X* CX •≰ X ۷ × **4**2 🖳 × -1.5 + 0.0 2.5 2.0 -3.0 0. 0.5 1.5 Nu/SQRT(Re)

Appendix C: Length Scale Data Sheets







Integra	l Ler	ngth Scale		<u>-</u>			
Lag		Corr.	Average	1	J =	8.658	m/s
	1	0.976	0.9505		req =	6000	
	2	0.925			ntegral =	0.070532	
	3					2.776866	in
	4	0.834	0.8195				,
	5 6	0.805 0.781	0.793 0.7705		Micro-length	Scale	
	7	0.76	0.7505		Lag 1 =	0.995	
					Freq =	15000	Hz
	9	0.723			U =	8.882	
	10				Micro =	0.008374	m
	11	0.69	0.683			0.329686	in
	12						
	13						
	14						
	15						
ļ	16						
<u> </u>	17 18						
-	19	+					
	20					· · · · · · · · · · · · · · · · · · ·	
	21						
	22						
	23						
	24						
	25						
	26						
	27						
	28						
<u> </u>	29			· · · · · · · · · · · · · · · · · · ·			
	30 31						
	32						
	33						
	34						
	35	0.424	0.4195				
	36						
	37						
	38						ļ
<u> </u>	39						
	40						
	41						
 	43						
 	4						
	45						
	46						
	47						
	48						
	49						
	_50						
<u> </u>	<u>51</u>						
-	52 53						
 	5						-
 	5 5						+
	- 5						
 	5						
	58						
	50						
	60	0.27	7 0.274				
	6	0.27	0.2675				

62	0.264	0.2615				
ස	0.259	0.2565				
64	0.254	0.252				
65	0.25	0.2485				
66	0.247	0.2455				
67	0.244	0.2425				
68	0.241	0.2395				
69	0.238	0.2365				
70	0.235	0.2335				
71	0.232	0.2305				
72	0.229	0.2275				
73	0.226	0.2245				
74	0.223	0.222				
75	0.221	0.2195				
76	0.218	0.217				
77	0.216	0.215				
78	0.214	0.213				
79	0.212	0.211				
80	0.21	0.2085				
81	0.207	0.2055	 			
82	0.204	0.202				
83	0.2	0.198				
84	0.196	0.195				
85	0.194	0.193				
86	0.192	0.191				
87	0.19	0.1895				
88	0.189	0.188				
89	0.187	0.186				
90	0.185	0.1835				
91	0.182	0.181				
92	0.18	0.179				
93	0.178	0.177				
94						
	0.176	0.175				-
95	0.174	0.1725				
96	0.171	0.169				
97	0.167	0.165				
98	0.163	0.1615				
99	0.16	0.159				
100	0.158	0.1565				
101	0.155	0.154				
102	0.153	0.1515				
103	0.15	0.1485				
104	0.147	0.146				
105	0.145	0.1435			<u> </u>	
106		0.141	 			
107	0.14	0.139				
108	0.138	0.139			ļ	
109	0.136	0.135			ļ	
110	0.134	0.133				
111	0.132	0.131				
112	0.13	0.129				
113	0.128	0.127		<u> </u>		
114	0.126	0.125				
115	0.124	0.123				
116	0.122	0.1215				
117	0.121	0.121				
118	0.121	0.121				<u> </u>
119		0.1205				
120	0.121	0.1200				
121	0.12	0.118				
122		0.116				
					 	-
123	0.115	0.1135				
124		0.1105				
125	0.109	0.107				
126	0.105	0.1035				

127 0.102 0.1 128 0.098 0.096	
128 0.098 0.096	
· · · · · · · · · · · · · · · · · · ·	
129 0.094 0.0925	
130 0.091 0.089	
131 0.087 0.086	
132 0.085 0.084	
133 0.083 0.082	
134 0.081 0.0805	
135 0.08 0.08	
136 0.08 0.0795	
137 0.079 0.079	
138 0.079 0.0785	
139. 0.078 0.0775	
140 0.077 0.076	
141 0.075 0.0735	
142 0.072 0.071	
143 0.07 0.0695	
144 0.069 0.0685	
145 0.068 0.0675	
146 0.067 0.0665	
147 0.066 0.0655	
148 0.065 0.065	
149 0.065 0.0655	
150 0.066 0.066	
151: 0.066 0.066	
152 0.066 0.0655	
153 0.065 0.0645	
154 0.064 0.0635	
155 0.063 0.0625	
156 0.062 0.061	
157 0.06 0.06	
158 0.06 0.0595	
159 0.059 0.0585	
160 0.058 0.058	
161 0.058 0.0575	
162 0.057 0.057	
163 0.057 0.0575	
164 0.058 0.059	
165 0.06 0.0605	
166 0.061 0.061	
167 0.061 0.06	
168 0.059 0.058	
169 0.057 0.056	
170 0.055 0.0545	
174 0.05 0.0495	
175 0.049 0.049	
176 0.049 0.0495	
177 0.05 0.0505	
178 0.051 0.0515	
179 0.052 0.052	
180 0.052 0.052	
181 0.052 0.0525	
182 0.053 0.0535	
183 0.054 0.0555	
184 0.057 0.058	
185 0.059 0.06	***************************************
186 0.061 0.062	
187 0.063 0.0635	
188 0.064 0.064	
189 0.064 0.0635	
190 0.063 0.062	
191 0.061 0.061	

2.375 in Grid

192	0.061	0.061				
193	0.061	0.061				
194	0.061	0.061				
195	0.061	0.0605				
196	0.06	0.0595				
197	0.059	0.0585				
198	0.058	0.057				
199	0.056	0.0555				<u> </u>
200	0.055	0.0545				
201	0.054	0.0535				
202	0.053	0.053				
203	0.053	0.0525				
204	0.052	0.0515			1	
205	0.051	0.05				
206	0.049	0.0485				
207	0.048	0.048				
208	0.048	0.0485				
209	0.049	0.049				
210	0.049	0.049				
211						
	0.049	0.049				
212	0.049	0.0485				
213	0.048	0.048				
214	0.048	0.047				
215	0.046	0.0455				
216	0.045	0.044				
217	0.043	0.042				
218	0.041	0.04				
219	0.039	0.038				
220	0.037	0.0355				
221	0.034	0.0325				
222			<u> </u>			
	0.031	0.03				
223	0.029	0.0275				
224	0.026	0.025				
225	0.024	0.023				
226	0.022	0.0215				
227	0.021	0.021				
228	0.021	0.021				
229	0.021	0.021				
230	0.021	0.0215				
231	0.022	0.0225				
232	0.023	0.023				
233	0.023	0.0225				
234	0.022	0.021				
235	0.022	0.019	 			
236	0.018					
		0.0175			ļ	ļ
237	0.017	0.0165				
238	0.016	0.0155				ļ
239	0.015	0.015		L		
240	0.015	0.0145				
241	0.014	0.014				
242	0.014	0.014				
243	0.014	0.0145				
244	0.015	0.0155				
245	0.016	0.016				
246	0.016	0.016				
247	0.016	0.0155				
248	0.015	0.0135				
249					ļ	
	0.014	0.0135				ļ
250	0.013	0.0135				
251	0.014	0.0145				
252	0.015	0.016				
253	0.017	0.0175				
254	0.018	0.0185				
255	0.019	0.0195			 	
256		0.02				
	0.02	U.UZ	1	l		1

257	0.02	0.0205				
258	0.021	0.0215				ĺ
259	0.022	0.0225				
260	0.023	0.022				
261	0.021	0.02				
262	0.019	0.018				
263	0.017	0.016				
264	0.015	0.015				
265	0.015	0.0155				
266	0.016	0.017				<u> </u>
267	0.018	0.017				
268	0.018	0.0205				
269	0.021	0.0215				
270	0.022	0.0215				
271	0.021	0.02				
272	0.019	0.0185				Ĺ
273	0.018	0.018				
274	0.018	0.018				
275	0.018	0.018				
276	0.018	0.0175				
277	0.017	0.017	7			
278	0.017	0.017				1
279	0.017	0.017				
280	0.017	0.0165				
281	0.016	0.016				
282	0.016	0.0155				
283	0.015	0.0155				
284	0.016	0.016				
285	0.016	0.016				
286	0.016					-
287		0.0155				
	0.015	0.015				
288	0.015	0.015				
289	0.015	0.015				
290	0.015	0.015				
291	0.015	0.015				
292	0.015	0.0155				ļ
293	0.016	0.016				
294	0.016	0.0155				
295	0.015	0.014				
296	0.013	0.0125				
297	0.012	0.012				
298	0.012	0.0125				
299	0.013	0.014				
300	0.015	0.0155				
301	0.016	0.016				
302	0.016	0.0165				
303	0.017	0.017				
304	0.017	0.017				
305	0.017	0.017				
306	0.017	0.017				<u> </u>
307	0.017	0.0165				
308	0.016	0.016				
309	0.016	0.016			 	
310	0.016	0.016				
311	0.016	0.0155				<u> </u>
312	0.015	0.0135				
313	0.014	0.0125				
314	0.011	0.01				
315	0.009	0.0075				
316	0.006	0.005				
317	0.004	0.0035				
318	0.003	0.002				
319	0.001	0.0005				

0.5 in Grid

ntogral Lo	noth Soolo				
	ngth Scale	A	• •	0.047	
Lag	Corr.	Average	 U =	9.347	
1	0.919	0.8435	Freq =	6000	
2	0.768		 Integral =	0.013199	
3	0.638	0.595		0.519636	in
4	0.552	0.526			
5	0.5				
6	0.465	0.4495	Micro-lengt		
7	0.434	0.419	Lag 1 =	0.985	
8	0.404	0.389	Freq =	15000	Hz
9	0.374	0.3605	U =	8.998	m/s
10	0.347	0.3345	Micro =	0.004898	m
11	0.322	0.309		0.19283	in
12	0.296	0.2835	-		
13	0.271	0.259			
14	0.247	0.236			
15					
16	0.205				
17			 		-
18	0.174				<u> </u>
19	0.16				
20	0.148		 		
21	0.137				
22			 		
23			 		
24			 -		
25			 		
26			 		
27			 		
28		<u> </u>	 		
				-	
29	!		 		
30			 		
31	0.054		 		
32					
33			 		
34			 		<u> </u>
35					
36	<u> </u>		 <u> </u>		
37					
38	·				
39					
40					
41	0.003	0.0025			
42	0.002	0.0025			
43	0.003	0.004			
44	0.005		 		
45					
46		<u> </u>			

Appendix D: Uncertainty Analysis

								_		
				% Uncertainty	q" Uncertainty	% q" Uncertainty	qc" Uncertainty	% qc" Uncertainty	h Uncertainty	% h Uncertainty
Current	5.00	5.00 +/- 0.0050 Amps	0 Amps	0.10	3.01	0.20	3.01	0.21	0.28	0.21
Resistance	2.51	-/- 0.125	2.51 +/- 0.1256 Ohms/sq	5.00	75.31	5.00	75.31		7.04	5.20
Film Width	0.20	0.20 +/- 0.0002 m	2 m	0.10	-2.93	-0.19	-2.93	-0.20	-0.27	-0.20
T inf	25.00 +	25.00 +/- 0.1500 deg C	0 deg C	09.0	Not Used	Not Used	0.77		1.97	1.45
TLC	35.70	35.70 +/- 0.1500 deg C	0 deg C	0.42	Not Used	Not Used	-0.85	90.0-	-1.98	-1.46
Emmisivity	0.85	0.85 +/- 0.1500	0	17.65	Not Used	Not Used	-10.16	-0.70	-0.95	-0.70
Conduction	00.0	·/- 0.000	0.00 +/- 0.0000 W/m^2	00.00	Not Used	Not Used	00.0		00.0	00.0
S-B Const	5.67E-08 W/(m^2*K^4)	V/(m^2*k	(v4)				+			
- I	1506 219 +/- 75 428 W/m^2	75.42	8 W/m^2	5.01	% I Incertainty					
#	1448 624 +/- 76 118 W/m^2	/- 76 11	8 W/m^2	5 25	5 25 % I Incertainty					
111111111111111111111111111111111111111	135 385 ±/-	7 54	7 641 W//m^2*K)	5.64	5.25 % Cilicontainty					
				% Uncertainty	q" Uncertainty	% q" Uncertainty	qc" Uncertainty	% qc" Uncertainty	h Uncertainty	% h Uncertainty
Current	2.00 +	2.00 +/- 0.0050 Amps	Amps	0.25	1.20	0.50			0.11	0.66
Resistance	2.51 +/-	/- 0.125t	0.1256 Ohms/sq	5.00	12.05	5.00	12.05		1.13	6.57
Film Width	0.20	0.20 +/- 0.0002 m	2 m	0.10		-0.19	-0.47		-0.04	-0.26
r inf	25.00 +	25.00 +/- 0.1500 deg C	C deg C	09.0	Not Used	Not Used	0.77	0.42	0.31	1.82
LLC	35.70 +	35.70 +/- 0.1500 deg C) deg C	0.42	Not Used	Not Used	-0.85		-0.32	-1.87
Emmisivity	0.85 +	0.85 +/- 0.1500		17.65	Not Used	Not Used	-10.16	-5.54	-0.95	-5.54
Conduction	+ 00.0	/- 0.000C	0.00 +/- 0.0000 W/m^2	00.00	Not Used	Not Used	00.0	00.0	00.0	00.00
S-B Const	5.67E-08 W/(m^2*K^4)	V/(m^2*K	^4)							
# 10	240.995 +/-	12.115	12.119 W/m^2	5.03	5.03 % Uncertainty					
" 2b	183.400 +/- 15.858 W/m^2	15.858	3 W/m^2	8.65	8.65 % Uncertainty					
-	17 140 11	L	1 E 4 4 18/11 - 10:41							